














OHMITE THERMAL PRODUCT GUIDE

	Heatsinkable	Part Number Prefix	Ω	Watts
	TAH/TBH/TCH	TAH20 / TAH25 / TCH35	0.05 - 10K	20 - 35
	TDH	TDH35	0.1 - 10K	35
	TEH	TEH100	10K	100
	TL	TL54 - TL122	0.5 - 51K	27 - 275
	TGHG	TGHG	0.0005 - 10K	100 - 200
	TAP600/TAP800	TAP600 / TAP800	1 - 10K	600 - 800
	TAP1000/TAP2000	TA1K0PH / TA2K0PH	0.50 - 1000	1000 - 2000
	BA	BA1-BA3	0.5 - 18K	500 - 1000
	IS	IS175 - IS270	5 - 10K	175 - 270
	MetalOhm	20/45M - 85/150M	0.05 - 100K	20 - 150
	89 Series	805 - 850	0.10 - 100K	5 - 50
	HS/HSN	HS100 - HS250	0.05 - 17.4K	100 - 250
	WFH	WFH90 - WFH330	0.22 - 39K	90 - 330

	Heat Sinks	Part Number Prefix	Compatible Packages	Thermal Resistance ¹ (C/W)
	C60/B60	B60 - C60	Multiple custom	0.7 - 2 in forced convection
	C40	C40	TO-247, TO264	1 - 3 in forced convection
	C Series	C126 - C264	TO-126, TO-220, TO247, TO-264	6.8 - 13.6
	CP4	CP4	TAP600/800, TAP1K0/2K0	0.015 - 0.03 at 0.5 to 2 gpm
	D	DA-T263, DV-T268	TO-252, TO-263, TO-268	4.2 - 6 with PCB conduction
	E	EV-T220, EA-T220	TO-220	6.2 - 11.4
	F and R	FA, RA	TO-218, TO-220, TO-247	3 - 5
	M	MA-102, MV302	TO-247, TO-264	7.5 - 15
	R2	R2V, R2A	TO-220, TO-247, TO-264	10
	S	SA-LED, SV-LED	STAR LED	2.5 - 6.6
	SV	SA-LED-3, SV-LED-3	LED Modules	5.6 - 9
	VM	VM1 - VM3	TO-220, TO-247	5
	W	WA-T220, WV-DT2	TO-220, TO-247, TO-264	15 - 18

1. Thermal resistance for 75°C mousing surface temperature rise in natural convection unless otherwise noted.



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Selecting a Heat Sink

1. Thermal Resistance Basics

T_c T_s T_a T_c : device case temperature (°C)

 T_s : heat sink temperature (°C)
 T_a : ambient temperature (°C)
 $R_{\theta cs}$ $R_{\theta sa}$
 Interface Resistance Heat Sink Resistance
 Q : heat dissipated (watts)
 $R_{\theta ta}$: thermal resistance (°C / W)

Basic thermal resistance equation

$$R_{\theta ca} = R_{\theta cs} + R_{\theta sa} = (T_c - T_a) / Q$$

Solving for the heat sink thermal resistance

$$R_{\theta sa} = ((T_c - T_a) / Q) - R_{\theta cs}$$

Or solving for heatsink temperature rise over ambient

$$(T_s - T_a) = (T_c - T_a) - (R_{\theta cs} \times Q)$$

2. Calculating the Heat Sink Requirement

Example

$$Q = 10 \text{ watts}$$

$$T_c = 150^\circ \text{ C max}$$

$$T_a = 40^\circ \text{ C max}$$

$$R_{\theta cs} = 1.0^\circ \text{ C / W}$$

(Varies with device package and interface material, such as thermal grease, silicone pad, Kapton, phase-change material, etc. Contact the factory for more details.)

Therefore, the thermal resistance required will be:

$$R_{\theta sa} = ((150 - 40) / 10) - 1 = 10^\circ \text{ C / W}$$

or lower value will be acceptable

Or, calculating the heat sink temperature rise:

$$(T_s - T_a) = (150 - 40) / (1 \times 10) = 100^\circ \text{ C}$$

or lower value will be acceptable (at 10 W power)

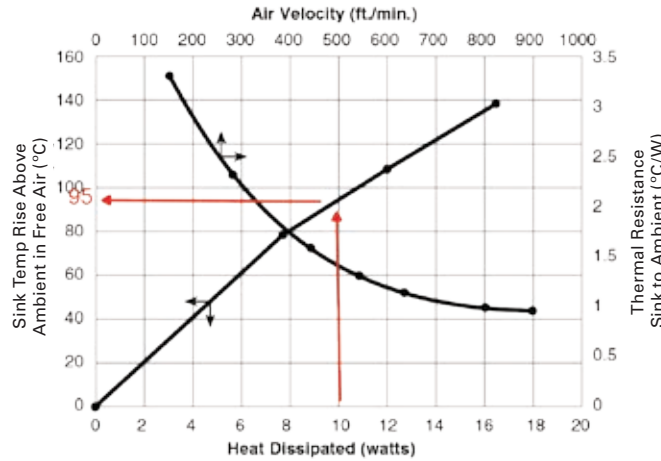
3. Selecting the Heat Sink

3a. Board Mounted Heat Sink

If it is a board (PCB) mounted heat sink, there will be a graph for the specific heat sink.

For the natural convection curve: power dissipation (watts) vs. heat sink temperature rise above ambient ($T_s - T_a$).

For the forced-convection curve: thermal resistance from mounting surface to ambient ($R_{\theta sa}$ (°C / W) vs. air velocity (ft. / min.)



For the specific example, ($T_s - T_a$) = 95° C at 10 watts power, so this heat sink would satisfy the thermal requirement in natural convection.

If the heat sink has a clip to hold the device, then the heat sink thermal data will be given in terms of case temperature, not sink temperature. Therefore, in natural convection the case temperature rise above ambient ($T_c - T_a$) will be plotted vs. heat dissipation. With known values of T_c max, T_a max, and Q (watts), one can go right to the graph for natural convection and determine whether the heat sink will work.

For forced convection, calculate the $R_{\theta ta}$ from the equation and go to the graph to determine the airspeed requirement.

$$R_{\theta ca} = (T_c - T_a) / Q$$

3b. Extrusion Heat Sink

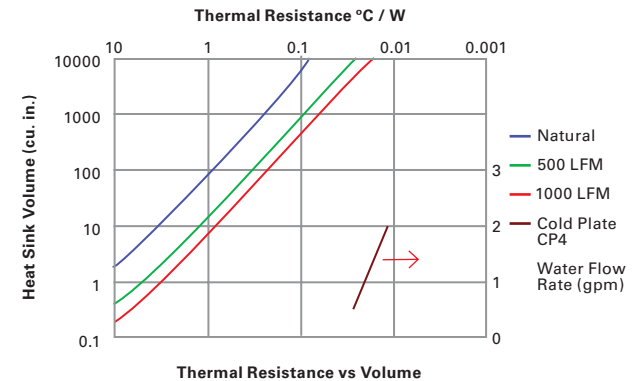
If the device has higher power dissipation and an extrusion is needed, use this quick sizing guide to find the approximate size (volume) of the heat sink to satisfy the thermal requirements. Then, using the data sheet of available Ohmite extrusions, one can select potential shapes and lengths that will meet or exceed this volume.

Example:

If the $R_{\theta sa}$ calculation requirement is 1.0° C / W, then from the chart the heat sink volume would be: For natural convection approximately 90 cubic inches or greater. For 500 ft./min. airspeed approximately 15 cubic inches or greater. 90 cubic inches could be satisfied by 9.5" length of extrusion AH13070 (6 x 1.75 x 9.5). 15 cubic inches could be satisfied by 3.6" length of extrusion AH12153 (3.6 x 1.15 x 3.6).

So, 9.5 inches of AH13070 is a starting point to select a heat sink for natural convection. 3.6 inches of AH12153 will work for 500 ft./min. of air flow. Interpolate for other air speeds. Other selections can be made based on available geometry of the space and any mounting considerations. However, this tool is only intended to be a first draft for selecting an extruded heat sink. For further thermal calculations and selection assistance contact the Ohmite Applications Engineering Desk.

The available Ohmite extrusions are found at www.ohmite.com/cat/sink_ah.pdf.



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